

IN THE SPECIFICATION

Please amend the specification by replacing paragraphs [0035], [0036], [0046], [0047], [0049], [0058], [0060], [0072], [0082], [0083], [0097], [0110] and [0119] with replacement paragraphs as follows:

[0035] $Fa = \frac{4*a*t}{S^2}$ ~~$\frac{4*a*t}{S^2}$~~ , where

[0036] $\alpha = \lambda/(p*c)$ ~~$\alpha = \lambda/(p*c)$~~ = the thermal diffusivity constant

[0038] $p = \text{density}$ ~~$\rho = \text{density}$~~

[0046] $\Delta T = \Delta T_0 \left(1 + 0.15 * F_0^{-1.9}\right) \left(1 + 0.15 * F_0^{-1.9}\right)$ (1)

[0047] $\Delta T_0 = \frac{2 * E}{(p * c * S)}$ ~~$\frac{2 * E}{(p * c * S)}$~~ , where

[0049] $p = \text{density}$ ~~$\rho = \text{density}$~~

[0058] where $t/t_0 = F_0$, t being = the duration of the thermal pulse, and $t_0 = \frac{S^2}{(4*a)}$ ~~$\frac{S^2}{(4*a)}$~~ being a constant characteristic of

[0060] $\Delta T / \Delta T_0 = 1/A$ (4)

[0072] From an initial temperature T_u (Figure 4), we have a surface temperature $T_u + \Delta T$ ~~$T_u + \Delta T$~~ on the brake disk immediately after braking. However, the temperature in the disk is evened out quickly to $T_u + \Delta T_0$, which represents the temperature at which the cooling process begins. The temperature difference between the brake disk and its environment when cooling begins is therefore $T_u + \Delta T_0 - T_k$, where T_k is the temperature of the cooling element. If the time until the next braking is t_n , we have the temperature T_n when the next braking begins.

[0082] Calculation of Remaining/Consumed Life: Figure 5 shows a relationship between maximum total temperature and number of braking cycles for wearing-out using log-log scales. The relationship consists of two linear functions [0] Q, P with different slopes. The reason why two curves are used is that the lining on the brake disk is broken down at high temperature and has a tendency to char. This is because, at high temperatures for linings made of paper, a chemical process, carbonization, takes place. The upper curve [0] Q, on the left in the figure, describes the strength in a brake disk, the lining of which has reached such a high temperature that charring has started.

[0083] The slope of the curves and the break-point between the upper curve [0] Q and the lower curve P are obtained from rig testing. The slope of the left, upper curve [0] Q may, however, be difficult to produce with great accuracy and, in such a case, it can be estimated with, for example, the Arrhenius function.

[0097] D is damage value per unit of time or distance (damage per hour or damage per kilometer), and_n 1 and n2 are the number of braking cycles per temperature level and unit of time or distance.

[0110] $St = \frac{dT \cdot a \cdot E}{(1-v)} \frac{dT \cdot \alpha \cdot E}{(1-v)}$

[0119] The following background documents are hereby expressly incorporated for purposes of disclosure in the present application, and for reference by concerned persons skilled in the relevant art:

[1] Lauster, E. and Staberoth, U. "Wametechnische Berechnungen bei Lamellenkupplungen" VDI-Z 115 (1973);

[2] Kruger, H. "Das Temperaturverhalten der nassen Lamellenkupplungen" Konstruktion 17 (1963);

[3] Tataiah, K. "An Analysis of Automatic Transmission Clutch-Plate Temperatures" SAE 720287;

[4] Roark, RaymondJ. "Formulas for stress and strain."